

# WEEK 9 SOL'NS

8.43 / Given: water in insulated piston/cyl.

State 1:  $m_1 = 0.1 \text{ kg}$   
 $T_1 = 100^\circ\text{C}$   
 $x = 0.90$

State 2:  $P_2 = 1.2 \text{ MPa}$

Find:  $W_{12}$

CV: cylinder

Soln: Assume 1. Adiabatic  $Q_{12} = 0$

2. Reversible Process

3. Neglect  $ke + pe$

4. Continuity  $m_1 = m_2 = m$

5. 1st Law:

$$\Delta E \approx \Delta U = Q - W$$

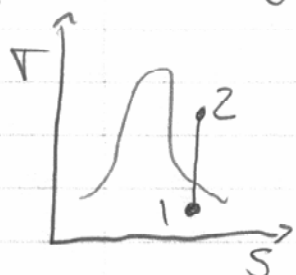
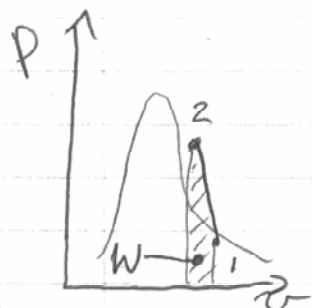
$$W_{12} = -m(u_2 - u_1)$$

6. 2nd Law:

$$dS = \int \frac{dQ}{T} = 0$$

$$m(s_2 - s_1) = \int \frac{dQ}{T} = 0$$

$$\therefore s_2 = s_1$$



State 1: from table B.1.1

$$s_1 = s_f + x s_{fg} = 1.3068 + 0.9(6.048)$$

$$s_1 = 6.75 \text{ kJ/kg K}$$

$$u_1 = u_f + x u_{fg} = 418.91 + 0.9(2087.88)$$

$$u_1 = 2297.7 \text{ kJ/kg}$$

State 2 from table B.1.3. (from  $P_2, s_2$ )

$$T_2 = 232.3^\circ\text{C}$$

$$u_2 = 2672.9$$

} By interpolation

$$\therefore W_{12} = -(0.1 \text{ kg})(2672.9 - 2297.7) \Rightarrow \boxed{W_{12} = -37.5 \text{ kJ}}$$

8.47/ Given: Ammonia in piston/cyl.

State 1:  $m_1 = 2 \text{ kg}$

$$T_1 = 50^\circ\text{C} = 323 \text{ K}$$

$$P_1 = 100 \text{ kPa}$$

State 2:  $P_2 = 1000 \text{ kPa}$

$$T_2 = T_1$$

Process: Reversible.

Find:  $Q_{12}$  and  $W_{12}$

Soln: CV, cylinder

Assume: 1. Reversible

2. neglect KE + PE

3. continuity  $m_1 = m_2 = m$

4. 1st Law:

$$\Delta E \approx \Delta U = Q_{12} - W_{12} = m(u_2 - u_1)$$

5. 2nd Law

$$dS = \int \frac{dQ}{T} = m(s_2 - s_1)$$

State 1 from table B.2.2

$$v_1 = 1.5658 \text{ m}^3/\text{kg}$$

$$u_1 = 1424.7 \text{ kJ/kg}$$

$$s_1 = 6.4943 \text{ kJ/kg K}$$

from table B.2.2

$$v_2 = 0.1450 \text{ m}^3/\text{kg}$$

$$u_2 = 1391.3 \text{ kJ/kg}$$

$$s_2 = 5.2654 \text{ kJ/kg K}$$

from 2nd Law  $Q_{12} = mT(s_2 - s_1) = 2(323)(5.2654 - 6.4943)$

$$Q_{12} = -794.2 \text{ kJ}$$

from 1st Law  $W_{12} = Q_{12} - m(u_2 - u_1) = -794.24 - 2(1391.3 - 1424.62)$

$$W_{12} = -727.6 \text{ kJ}$$

8.59 / Given: Steam in rigid, insulated tank (Transient Process)  
 Overall process: irreversible  
 but steam remaining in tank goes through adiabatic +  
 Reversible expansion. (inside the tank)

State 1  $P_1 = 3 \text{ MPa}$       State 2  $x_2 = 1$   
 $T_1 = 400^\circ\text{C}$

Find: fraction of steam that escapes,  $\frac{m_e}{m_1}$

Soln: CV, steam inside tank

Assumptions 1. Reversible

2. Adiabatic  $Q_{12} = 0$

3. Continuity  $m_e = m_1 - m_2$

4. 1st Law:

$$\Delta E \approx \Delta U = Q_{12} - W_{12} = m(u_2 - u_1)$$

5. 2nd Law

$$\frac{dS}{dt} = \int \frac{dQ}{T} \quad (\text{for reversible process})$$

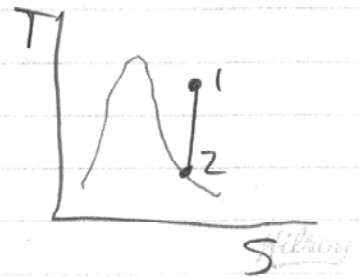
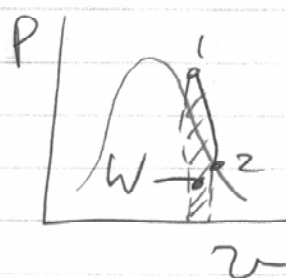
$$m(S_2 - S_1) = 0 \quad \therefore S_2 = S_1$$

$S_1 = S_2 = 6.9212$  From  $P_1, T_1$  in superheated tables

$\therefore T_2 = 141^\circ\text{C}$  from  $S_2, x_2$  in table B.1.1

$$\therefore \frac{m_e}{m_1} = \frac{m_1 - m_2}{m_1} = 1 - \frac{m_2}{m_1} = 1 - \frac{v_2}{v_1} = 1 - \frac{0.09936}{0.4972}$$

$$\boxed{\frac{m_e}{m_1} = 0.80}$$



8.60/ Given: water in Piston/cyl.

State 1:  $m_1 = 2 \text{ kg}$   
 $T_1 = 200^\circ\text{C}$   
 $P_1 = 10 \text{ MPa}$

State 2:  $P_2 = 200 \text{ kPa}$   
 $T_2 = T_1$

Ambient:  $T_0 = 200^\circ\text{C}$

Process: isothermal  $T_2 = T_1$

Find:  $Q_{12} + W_{12}$

Sol'n C.V. water

Assume: 1. Continuity  $m_1 = m_2 = m$

2. neglect ke + pe

3. reversible process

4. 1st Law:  $\Delta E \approx \Delta U = Q_{12} - W_{12} = m(u_2 - u_1)$

5. 2nd Law:  $dS = \frac{dQ}{T}$

$$mT(s_2 - s_1) = Q_{12}$$

State 1: table B.1.4  $\Rightarrow$  (compressed liquid table)

$$v_1 = 0.001148 \text{ m}^3/\text{kg}$$

$$u_1 = 844.49 \text{ kJ/kg}$$

$$s_1 = 2.3178 \text{ kJ/kg K}$$

$$V_1 = m v_1 = (2 \text{ kg})(0.001148 \text{ m}^3/\text{kg}) = 0.0023 \text{ m}^3$$

State 2: table B.1.3  $\Rightarrow$  (super heated)

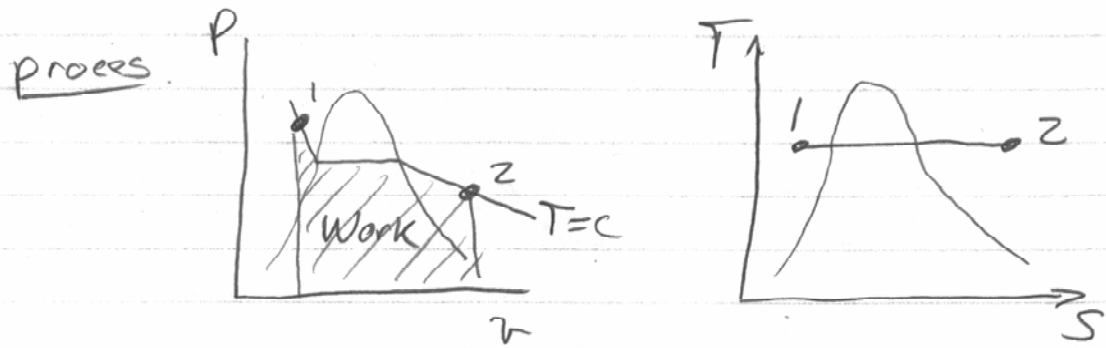
$$v_2 = 1.08034 \text{ m}^3/\text{kg} \quad u_2 = 2654.4 \text{ kJ/kg}$$

$$s_2 = 7.5066 \text{ kJ/kg K}$$

$$V_2 = m v_2 = (2 \text{ kg})(1.08034 \text{ m}^3/\text{kg}) = 2.1607 \text{ m}^3$$

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8.60 cont...



from 2nd Law

$$Q_{12} = mT(s_2 - s_1) = 2(473)(7.5066 - 2.3178)$$

$$Q_{12} = 4910 \text{ kJ}$$

from 1st Law  $W_{12} = Q_{12} - m(u_2 - u_1)$

$$W_{12} = 1290.3 \text{ kJ}$$

8.126/ Given:  $\text{H}_2\text{O}$  in insulated cyl./piston

state 1:  $V_1 = 0.15 \text{ m}^3$   
 $P_1 = 400 \text{ kPa}$   
 $T_1 = 200^\circ\text{C}$

state 2: ?

Process: adiabatic,  $Q_{12} = 0$ ,  $W_{12} = 30 \text{ kJ}$

Find: if state 2 is in two-phase region.

Sol'n C.V.  $\text{H}_2\text{O}$

- Assume
1. Continuity  $m_1 = m_2 = m$
  2. Reversible process

8.126  
cont....

3. 1st Law:  $\Delta E \approx \Delta U = m(u_2 - u_1) = Q_{12} - W_{12}$   
 $W_{12} = -m(u_2 - u_1)$

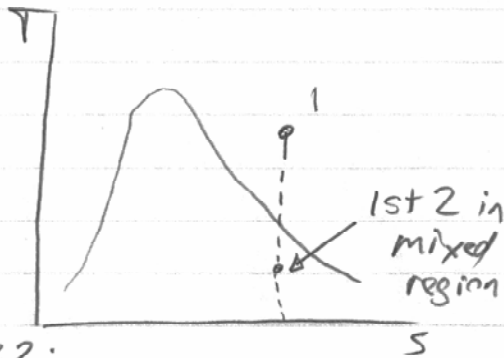
4. 2nd Law:  $dS = \int \frac{dQ}{T}$

$$m(s_2 - s_1) = 0 \Rightarrow s_2 = s_1$$

State 1: from table B.1.3 (superheated)

for  $T_1, P_1 \Rightarrow v_1 = 0.5342 \text{ m}^3/\text{kg}$   $u_1 = 2646.8 \text{ kJ/kg}$

$$s_1 = 7.1706 \text{ kJ/kgK} = s_2$$



$$m = \frac{V_1}{v_1} = \frac{0.15}{0.5342} = 0.2808 \text{ kg}$$

State 2:

from 1st Law:  $u_2 = u_1 - \frac{W_{12}}{m} = 2646.8 - \frac{30}{0.2808}$

$$u_2 = 2540 \text{ kJ/kg}$$

from table B.1.1, if  $u_2 = u_g$

$$s_g = 7.0259$$

$$s_g < s_2 \therefore u_2 \neq u_g$$

State 2 must be in superheated region

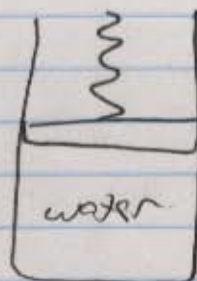
Given: water in piston/cyl.

8.132/

State 1:  $m_1 = 3 \text{ kg}$   
 $P_1 = 500 \text{ kPa}$   
 $T_1 = 600^\circ\text{C}$

State 2:  $T_2 = 20^\circ\text{C}$

piston  $A = 0.1 \text{ m}^2$   
spring  $K = 10 \text{ kN/m}$



Find:  $\Delta S_{\text{gen}}$

Soln: Assume 1. continuity:  $m_1 = m_2 = m$   
2. 1st Law:  $\Delta E \approx \Delta U = Q_{12} - W_{12} = m(u_2 - u_1)$   
3. 2nd Law:

$$dS = \frac{dQ}{T} + dS_{\text{gen}}$$

$$\text{or, } \Delta S_{\text{gen}} = m(s_2 - s_1) - \frac{Q_{12}}{T}$$

State 1

from table B.1.3  $\Rightarrow$   $v_1 = 0.8041$   
 $u_1 = 3299.6$   
 $s_1 = 7.3522$

State 2: from force balance at piston

$$P = P_1 + \left(\frac{K_s}{A^2}\right)(V - V_1)$$

$$\text{since } \sum F_y = 0 = P_1 A - P_0 A - K_s x_1 \quad \text{at state 1}$$

$$\text{or } 0 = P_1 A - P_0 A - \frac{K_s V_1}{A}$$

$$\text{and } \sum F_y = 0 = P A - P_0 A - \frac{K_s V}{A} \quad \text{at any point}$$

$$\therefore P_1 A - P_0 A - \frac{K_s V_1}{A} = P A - P_0 A - \frac{K_s V}{A}$$

$$\text{or } P = P_1 + \left(\frac{K_s}{A^2}\right)(V - V_1)$$

- Assume state two is in two phase region.

$$\therefore P_2 = P_{\text{sat}} @ (T_2) = 2.339 \text{ kPa}$$

$$\therefore v_2 = v_1 + \frac{(P_2 - P_1) A^2}{m K}$$

$$v_2 = 0.8041 + \frac{(2.339 - 500)(0.01)}{3(10)} = 0.6382$$

$$v_2 = v_f + x_2 v_{fg}$$

yes, in two phase region

$$\therefore x_2 = \frac{(0.6382 - 0.001002)}{57.7887} = 0.011$$

$$\therefore u_2 = 109.46, s_2 = 0.3887$$

~~from first law~~

$$\text{Work: } W_{12} = P_{\text{avg}} \cdot m (v_2 - v_1) = \frac{1}{2} (500 + 2.339) 3 (0.6382 - 0.8041)$$

$$W_{12} = -125 \text{ kJ}$$

from ~~second~~ first Law

$$Q_{12} = m(u_2 - u_1) + W_{12} = 3(109.46 - 3299.6) - 125$$

$$Q_{12} = -9695.4 \text{ kJ}$$

from Second Law

$$\Delta S_{\text{gen}} = m(s_2 - s_1) - \frac{Q_{12}}{T_{\text{room}}}$$

$$\Delta S_{\text{gen}} = (3)(0.3887 - 7.3522) + \frac{9695.4}{293.15}$$

$$\Delta S_{\text{gen}} = 12.18 \text{ kJ/K}$$

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8.135/

Given: air in piston/cyl.State 1:  $P_1 = 500 \text{ kPa}$ 

$$T_1 = 200^\circ\text{C} = 473 \text{ K}$$

$$V_1 = 10 \text{ L} = 0.01 \text{ m}^3$$

State 2:  $P_2 = 150 \text{ kPa}$ 

$$V_2 = 25 \text{ L} = 0.025 \text{ m}^3$$

Ambient:  $T_0 = 20^\circ\text{C} = 293 \text{ K}$ ProcessSoln: 70% of work is reversible, adiabatic

b) process is

Find: a)  $W_{12}$  and if it's possible.Soln: C.V. Air,  $R = 0.287 \text{ kJ/kg K}$ ,  $C_p = 1.004 \text{ kJ/kg K}$   
 $C_v = 0.717 \text{ kJ/kg K}$ 

State 1:  $m_1 = \frac{P_1 V_1}{R T_1}$  (assume Ideal Gas)

$$m_1 = 0.0368 \text{ kg}$$

Process isentropicSoln: efficiency of process  $\eta_s = 0.70$   
(actual work is 70% of isentropic work)

a) Assume reversible and adiabatic process

∴ from second law

$$dS = \frac{dQ}{T} \text{ or } m(s_{2s} - s_1) = 0$$

$$\therefore s_{2s} = s_1$$

for a polytropic process,  $PV^n = \text{constant}$   
and has the relation,

$$\frac{T_2}{T_1} = \left(\frac{P_2}{P_1}\right)^{\frac{n-1}{n}} = \left(\frac{V_1}{V_2}\right)^{n-1} \Rightarrow \therefore T_{2s} = T_1 \left(\frac{P_2}{P_1}\right)^{\frac{n-1}{n}}$$

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$n = k$  for an <sup>isentropic</sup> ~~isothermal~~ process (p. 279 in thermo book)

for a reversible process,

$$T_{2s} = T_1 \left( \frac{P_2}{P_1} \right)^{\frac{k-1}{k}} = 473 \left( \frac{150}{500} \right) = 335 \text{ K}$$

from first law:  $Q_{12} = m(u_{2s} - u_1) + W_{12}$   
 $Q_{12} = 0$

Assume constant specific heats (ideal gas)

$$W_{12} = m C_v (T_1 - T_{2s}) = 3.63 \text{ kJ}$$

$$\therefore W_{12} (\text{actual}) = \eta W_{12} = (0.7)(3.63)$$

$$W_{12} (\text{actual}) = 2.54 \text{ kJ}$$

b) from ideal gas law:  $T_2 = T_1 \frac{P_2 V_2}{P_1 V_1} = 354.9 \text{ K}$

1st law:  $Q_{12} = m C_v (T_{2(\text{actual})} - T_1) + W_{12} (\text{actual})$   
 $Q_{12} = -0.58 \text{ kJ}$

2nd law:  $\Delta S_{\text{total}} = m(s_2 - s_1) - \frac{Q_{\text{cv}}}{T_0}$ ,  $Q_{\text{cv}} = Q_{12} (\text{actual})$

$$m s_2 - s_1 = C_p \ln \frac{T_2}{T_1} - R \ln \frac{P_2}{P_1} = 0.0569 \text{ kJ/kg K}$$

$$\Delta S_{\text{total}} = 0.00406 \text{ kJ/K} > 0$$

$\therefore$  process is possible.